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Review article

The development of radiant floor cooling system (RFCS): System type, control strategy, and application

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ABSTRACT

The utilization of radiant floor systems is one effective method to address the increasing cooling demand. The radiant floor cooling system (RFCS) has self-regulating capabilities, which benefit grid stability and the use of renewable energy. The application performance of RFCS is based on system design and intelligent control, as systems with high thermal inertia make it difficult to provide ideal indoor response effects. Therefore, this review aims to summarize the current development status of RFCSs and evaluate future development directions based on relevant research trends. A comprehensive overview of the radiant floor structure and the composition of RFCS were provided. The type of radiant floor structures can bring greater possibilities for the application of RFCS. Then, system control and evaluation are analyzed, with research on anti-condensation of radiant floors mainly focused on system and environmental control. Based on application status of the RFCS, the impact of various factors is discussed, including condensation risks, and climate change. Resistance to disturbances not only requires implementing control measures but also integrating with the building environment. There is still great potential for the hardware design and multi-directional integration of RFCS. The results of this review help relevant researchers gain a comprehensive understanding of the current development status and prospects of RFCS contributing to the decarbonization of the construction industry.

1. Introduction

In recent years, the construction industry has gradually veered off track to meet the Paris targets, with construction operations accounting for around 30 % of global energy sector emissions in 2022 [[1](#page-18-0)]. To combat this, the building industry needs to promote zero-carbon, resource-efficient, and resilient buildings [[2](#page-18-0)]. Radiant cooling systems can integrate into zero-energy or green buildings while providing equivalent or superior comfort compared to all-air systems [3[–](#page-18-0)5]. According to the types of radiant cooling terminal, the system can be divided into radiant floor system, radiant wall system, and radiant ceiling system. Among these, radiant floor cooling system (RFCS) has been extensively utilized in engineering for efficient cooling [\[6,7](#page-18-0)].

Radiant floor cooling system is an evolution of radiant heating

systems. Initially, people used hot gases or water to raise indoor temperature by heating the floors, walls, and kang. The success of this heating method gained the interest of researchers, leading to the development of radiant floor heating systems and the subsequent popularity of this system. With the demand for indoor thermal comfort and energy-saving performance improved, researchers began to explore radiant cooling in the 1930s [[8](#page-18-0)]. These systems have been successfully implemented in buildings, marking a new chapter in radiant floor cooling. The radiant floor system has advantages in indoor cooling, despite their low heat transfer coefficient [[9](#page-18-0)]. Energy saving is one of the advantages of RFCS. The larger heat exchange area of the radiant floor allows it to regulate the indoor environment under smaller temperature differences between radiant floor temperature and air temperature. High temperature cooling water can be obtained through natural cold

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sources in order to reduce the use of primary energy $[10,11]$ $[10,11]$.

As a low-exergy technology, RFCS can maximize the coefficient of performance in the built environment by reducing exergy demand and increasing exergy efficiency $[12,13]$ $[12,13]$. To prevent condensation on the radiating surface, the radiant floor surface temperature cannot exceed the indoor dew point temperature [[14\]](#page-18-0). While this limits cooling ability of the radiant floor, RFCS could transfer indoor cooling loads through the thermal mass of the building, reducing peak loads [\[15](#page-18-0),[16\]](#page-18-0). However, high thermal mass will result in long system response times [[17,18](#page-18-0)], which is not conducive to the transient control of indoor thermal environment. Furthermore, inefficient utilization of energy stored within buildings can lead to energy wastage [[19,20\]](#page-18-0). This problem can be addressed by implementing flexible control strategies and suitable system types [\[21](#page-18-0),[22\]](#page-18-0). The operating performance of RFCS is affected by the climate parameters and building types. Specifically, the system can bear more indoor cooling load in dry climates compared to wet ones [\[23](#page-18-0)]. To improve the adaptability of RFCS in different climate zones, coupling ventilation system or dehumidification system is necessary to cope with indoor latent load or auxiliary cooling [\[24](#page-18-0)–26]. The energy consumption of the integrated system is inversely proportional to the sensible load that the ventilation system bears [\[27](#page-19-0)]. Ultimately, the operational performance and energy efficiency of the hybrid system are significantly affected by the system design and control strategy.

Previous reviews have extensively covered various aspects of the RFCS, as summarized in Fig. 1. They mainly focus on "system design" [[26,28,29](#page-19-0)], "thermal comfort analysis" [\[3](#page-18-0)[,30,31](#page-19-0)], "system control" [\[32](#page-19-0), [33\]](#page-19-0), "economic analysis" [[34,35](#page-19-0)], "system application" [\[36,37](#page-19-0)], "heat transfer and model" [[38\]](#page-19-0) and "cooling capacity" [[10\]](#page-18-0). Although this review intersects with previously discussed topics in the literature, they have primarily focused on the entire radiant system. According to current literature, there are significant differences between various types of radiation terminals, including condensation risk, cooling efficiency, and energy-saving benefits. The theories and application guidelines related to different types of radiation systems are not universally applicable. Although there has been independent research on the system, control, and parameter analysis of radiant floor, there is a lack of organization and exploration of various technological developments. Therefore, it is necessary to make further efforts to review the opportunities and

challenges of RFCS.

[Fig.](#page-2-0) 2 shows the sustainable development methods for implementing the system. Based on this, this article first discusses different types of systems, including radiant floor systems and ventilation systems, and studies the advantages and limitations of integrated systems. Then, the focus was on introducing the relevant theoretical knowledge and design criteria of RFCS in control. Combined with climate and architectural characteristics to reduce fluctuations caused by uncertain factors, the application challenges were analyzed. Finally, a conclusion was drawn and the future development of RFCS was discussed to highlight its competitiveness in ultra-low energy buildings.

2. System type

The radiant floor will be integrated into the building, and the purpose of system design is to combine subjective and objective conditions to initially reduce system energy consumption and ensure thermal comfort. Reasonable design of different subsystems is crucial for the response rate and control effectiveness of subsequent systems. In this section, a comprehensive introduction was given to three parts: radiant floor, ventilation systems, and cold sources, and the latest research progress in system design was summarized.

2.1. Radiant floor structures and cold sources

2.1.1. Radiant floor structures

The hydraulic radiant floor structure mainly consists of floor covering (Tiles, marble, and concrete, etc.), weight-bearing (cement screed, anhydrite screed, asphalt screed, and timber), and thermal diffusion layer (aluminum heat diffusion devices, and Steel heat diffusion devices), insulating insulation (Polystyrene foam and foamed concrete, etc.), acoustic insulation (if present), structural bearing, contraction joints, expansion joints (movement joints), and construction joints composition [[39\]](#page-19-0). Based on the thermal response time, radiant floors can be classified into two categories: medium-response radiant floors and slow-response radiant floors [[18\]](#page-18-0), as illustrated in [Fig.](#page-2-0) 3. The main difference between these two types lies in the placement of the buried pipe and the presence of insulation. The most common type of

Fig. 1. The research points of RFCS mentioned in the previous review literature.

Fig. 2. The sustainable development methods for implementing the system.

Fig. 3. Structural representation of typical radiant floors [[18\]](#page-18-0): (a) Medium-response radiant floor and (b) Slow-response radiant floor.

radiant floor is the concrete core radiant floor, in which pipes are embedded within the concrete layer [[40\]](#page-19-0). Pipes pass through the concrete slabs to deliver cooling to the interior and store in concrete slab.

The thermal capacity of the system is primarily determined by the thickness of the surface layer in which the pipes are embedded [\[41](#page-19-0)]. A slow-response radiant floor possesses a larger heat storage capacity that can minimize the impact of indoor load and external climate on the indoor thermal environment. When turned on and off, charging and discharging phenomena occur. Medium-response radiant floor exhibits good insulation and has lower thermal mass, leading to lower thermal inertia and faster reaction times for floor. When the step change in control of the system is applied as input, it takes 9.7–18.8 h for the surface temperature of the slow-response radiant floor system to reach 95 % of the difference between its final and initial values, while the medium-response takes 2.3–8.7 h. Due to the reduction in the cooling of the building envelope, Medium-response radiant systems are more suitable for retrofitting existing buildings than slow-response radiant floor. The cooling capacity of radiant floor demonstrates similar tends across different response times [[42\]](#page-19-0). Most simulation studies presume steady-state conditions.

According to the methods of construction and installation of hydraulic radiant floor, the hydraulic radiant floor can be categorized into two types: wet construction (also known as heavyweight system) and dry construction (also known as lightweight system) [[43\]](#page-19-0), as shown in [Fig.](#page-3-0) 4. In wet construction, pipes are covered with cement or mortar and embedded directly into the building structure. The temperature distribution on the surface of a floor constructed using the wet method is more uniform [[44\]](#page-19-0). However, with advancements in technology, dry construction has become more popular. The pipes in dry construction are placed on a heat insulation layer, making them easy to install and maintain. They are particularly useful for secondary air conditioning system transformations [[45,46\]](#page-19-0). Dry construction radiant floor has smaller thermal resistance, which has prompted most research to focus on their dynamic performance. Thomas et al. [\[47](#page-19-0)] established a finite

Fig. 4. Two pavement methods of radiant floor [\[45](#page-19-0)]: (a) Wet construction and (b) Dry construction.

element numerical model of the dry construction radiant floor and tested the accuracy of the model through experiments. It was found that the energy-saving advantage of light floor in steady-state mode was not significant compared to classical radiant floor. Zhao et al. [\[22](#page-18-0)] compared the dynamic behavior of heavyweight system and lightweight system and found that heavyweight floor when stably operated in high-intensity solar radiation environments, exhibited better performance than lightweight floors, with a cooling capacity 3.06–5.73 W/m^2 higher than that of low thermal capacity floors. They also proposed a method for predicting peak cooling loads.

Active hollow core slabs developed and popular in Europe that utilize air as a heat transfer medium are also used in building cooling [\[48](#page-19-0)]. Fig. 5 displayed the basic structure of active hollow core slabs, which employ fans to circulate air through hollow cores, enabling a cycle of charging, storage, and discharging. Underfloor ventilation system serves two primary functions: cooling the air in the pipe using the low-temperature floor, and providing radiative cooling to the room by blowing cold air [[48](#page-19-0),[49\]](#page-19-0). However, due to the limited air-to-floor contact area, it is necessary to improve the composition of the radiant floor to enhance its cold storage capacity and make up for the insufficient cooling supply. Active hollow core slabs necessitate sufficient ventilation space in the floor structure. Hydraulic radiant floors are more widely used and have unique advantages than active hollow core slabs. Therefore, the review pays more attention to the water medium radiant floor.

Based on the existing research, the radiant floor structure and material aspects have been optimized, as presented in [Table](#page-4-0) 1. The optimization goals primarily focus on the response capacity and cold storage capacity [[43,50](#page-19-0)]. An uneven temperature on the surface of the radiant floor can lead to thermal discomfort and localized condensation. By incorporating a steel base into the radiant floor, the uneven surface temperature can be reduced, subsequently affecting the heat storage capacity. Despite the potential for longer response times, energy storage can enhance the cooling capacity of the floor and reduce peak load [\[51](#page-19-0), [52\]](#page-19-0). From the table, it can be seen that using phase change materials to improve the cold storage capacity of radiant floors has become a research hotspot. The design of radiant floor should not only consider performance but also factors such as installation flexibility and cost.

2.1.2. Cold sources

The indoor environmental parameters and indoor thermal comfort affect the water supply temperature of the system, which is generally between 16 and 22 ℃ [\[61\]](#page-19-0). Sometimes lower temperatures have been chosen in certain studies [[62\]](#page-19-0). Thus, RFCS offers more flexibility in selecting the cold source. Typically, traditional air conditioning systems rely on chillers to produce cold water at around 7 ◦C to meet air cooling and dehumidification requirements. However, when using a chiller as the cold source of radiant floor system, it is necessary to mix high-temperature water or use heat exchanger to adjust the supply water temperature [\[63](#page-19-0)]. To obtain high-temperature cooling water, alternative cold sources such as heat pumps [\[64](#page-19-0),[65\]](#page-19-0), solar cooling systems [\[66](#page-19-0)], evaporative cooling systems [\[67,68](#page-19-0)], and direct-ground cooling systems [[12\]](#page-18-0) are preferred. These advanced technologies have gained popularity due to their applicability to radiant floor systems.

Heat pumps utilize a small amount of high-level energy to transfer heat from a low-level heat source to a high-level heat source, with a high coefficient of performance (COP). The incorporation of radiant floor system can improve the COP of heat pumps and reduce the demand for primary energy. Under the same power generation source, the radiant floor systems using heat pumps have shown promise as energy-saving systems that reduce carbon emissions and exergy loss [[69\]](#page-19-0). The COP of the geothermal heat pump and radiant floor system can achieve 5.29, effectively fulfilling the cooling demand of the building [[70\]](#page-19-0).

However, the radiant floor systems have strong adaptability to direct cooling from natural cold sources and the hybrid systems are more environmentally-friendly and energy-efficient [[63,71](#page-19-0)]. The use of natural cold sources comes with strict outdoor requirements. The temperature of water through the direct-ground cooling system and evaporative cooling system can fluctuate based on seasonal weather and other operational factors [\[72](#page-19-0)]. Improper design of cooling requirements may result in soil energy imbalance or indoor overheating [[73,74](#page-19-0)]. Therefore, natural cold sources are commonly used as an auxiliary source of the system. In fact, due to radiant floor energy storage, natural cold sources can also provide considerable hours of acceptable thermal comfort for the room in extreme climates [[68,75\]](#page-19-0). [Fig.](#page-5-0) 6 displays the schematic diagram of the RFCS with direct-ground cooling system employed in an office building [[11,](#page-18-0)[76](#page-20-0)]. The use of geothermal heat exchangers as a cold source provided good thermal comfort without requiring additional steam compression cycles [[12\]](#page-18-0). Hybrid renewable

Fig. 5. Structure of active hollow core slabs [[48](#page-19-0)].

Table 1

Summary of research on radiant floor structure.

energy systems can help alleviate energy intermittency issues [[77\]](#page-20-0). The COP of a radiant floor system that integrates ground temperature, water evaporation, and sky radiation can reach 19 during operation [[78\]](#page-20-0). City tap water can also be used as a source of cooling for radiant floor systems in residential buildings [[79\]](#page-20-0).

2.2. Combination with air system

The air system is crucial for the radiant floor system to operate effectively in hot and humid environments. Compared to the two, hybrid systems composed of both can combine their respective advantages and

compensate for their shortcomings. The following provides an overview of different hybrid system studies and types of ventilation systems are introduced below.

2.2.1. System characteristic

Radiant floor system is inefficient as a standalone method in hot and humid climates due to the risk of condensation on the floor surface. To improve cooling capability of system, natural night ventilation cooling is commonly employed as an auxiliary cooling method in conjunction with radiant floor systems [[65,](#page-19-0)[80\]](#page-20-0). However, the enhancements provided by natural ventilation alone are insufficient. It is common to integrate them

Fig. 6. The cold source type of the radiant floor system. (a) Schematic diagram of the direct-ground cooling system with RFCS [[11\]](#page-18-0), and (b) Schematic drawing of direct-ground cold source system in summer [\[76](#page-20-0)].

with convection systems in buildings (see [Fig.](#page-6-0) 7), such as cooling systems (dry coil units, chilled beams) [\[81,82](#page-20-0)], dehumidification systems (dehumidifiers) [[83\]](#page-20-0), and cooling and dehumidification (wet coil units, air handler units (AHU), variable air volume (VAV)) [[50,](#page-19-0)[84,85](#page-20-0)]. These integrated convection systems offer a wider operating range and quick response to changes in the indoor environment while controlling indoor humidity. The optimal cooling proportion of radiant floor system in integrated system with large spaces is 0.76–0.77 [[86\]](#page-20-0).

Ventilation systems help to enhance the cooling capacity of integrated systems and improve the response capabilities. Cooling and dehumidification systems are the most widely used to treat outdoor

fresh air directly. Zarrella et al. [\[25](#page-18-0)] conducted a comprehensive study to compare five types of RFCSs used in residential buildings. The study revealed that, when indoor conditions were maintained constant, fan coil and mechanical ventilation provided better thermal comfort than the dehumidification system. Moreover, mechanical ventilation can improve indoor air quality. Radiant floor system integrated with separate cooling and dehumidification systems allows for a larger range of temperature fluctuations in the radiation floor. Gu et al. [\[64](#page-19-0)] investigated a hybrid radiant floor system that integrated a fan coil unit and an outdoor fresh air dehumidifier. Their findings indicate that the fan coil assists the radiant floor system in providing cooling and adjusting the

Fig. 7. Classification of integrated convective systems for radiant floor cooling systems.

environment in different areas, while the outdoor fresh air dehumidifier controls indoor air humidity to maintain air quality. Overall, a separate fan coil in hybrid system can not only achieve satisfactory indoor thermal comfort but also achieve multi-zone control. On the other hand, dehumidification systems which integrate solar heat and indoor air heat recovery reduce the moisture content in the air, making them suitable for room with high latent heat loads or where maintaining low indoor humidity levels is necessary [\[87](#page-20-0)]. If only considering the removal of the typical indoor latent load, the exergy efficiency of the condensing dehumidification system can be 3 to 4 times higher than that of the desiccant wheel dehumidification system [[88\]](#page-20-0).

2.2.2. Ventilation types

Anti-condensation control on radiant floor surfaces and maintaining indoor air quality are essential responsibilities of the ventilation system. Integrated systems help to maximize the efficiency of the cooling system [[83\]](#page-20-0) and make the radiant floor system adaptable to different climates [[23](#page-18-0)[,65](#page-19-0)]. When radiant floor systems have large thermal mass, it can improve the response time of hybrid system. Compared to traditional convective air conditioning, the combination of a ventilation system and a radiant floor system reduces the cooling energy required for ventilation. This is due to the lower air mass flow rate, which results in reduced ventilation system noise and a smaller ventilation pipeline volume [\[31](#page-19-0)]. Fig. 8 shows a system diagram of RFCS with different ventilation types. The ventilation methods for RFCS include displacement ventilation [[86\]](#page-20-0), mixing ventilation [\[89](#page-20-0)], personalized ventilation [\[90](#page-20-0)], and underfloor ventilation [\[91,92](#page-20-0)].

Displacement ventilation offers strong sewage discharge capability and has low operating costs. The combination of radiant floor system and displacement ventilation system has a direct and positive effect on the indoor thermal environment, and the presence of radiant floors aids heat transfer [[93\]](#page-20-0). The ventilation vents near the floor effectively isolate the indoor humid air from the floor and reduce the risk of condensation on radiant surfaces. Two potential issues arise: the effect of supply air temperature on the heat transfer of the radiant floor and the distribution of indoor temperature gradients when vents close to the floor. However,

Fig. 8. Schematic diagram of radiant floor system and different ventilation methods [\[31\]](#page-19-0): (a) displacement ventilation (DV) + radiant panel, (b) mixing ventilation (MV) + radiant panel, (c) personalized ventilation (PV) + radiant panel, and (d) underfloor ventilation (UFAD) + radiant panel.

it does not cause thermal discomfort on the feet/calves [[94,95\]](#page-20-0). Schellen et al. [\[96](#page-20-0)] found that the vertical temperature gradients in the room with radiant floor system combined displacement ventilation could up to 4 ◦C and provides better comfortable conditions than combined mixing ventilation. In high-space buildings, the combination of displacement ventilation and radiant floor system allows for thermal comfort adjustment solely in the occupied space [[62\]](#page-19-0). The radiant floor system integrated displacement ventilation system has been successfully applied to the departure hall and check-in hall of Terminal 3 of Xi'an Xianyang International Airport. This system provides good thermal comfort during cooling periods, as well as a 34 % reduction in energy consumption compared to traditional jet ventilation systems [[83\]](#page-20-0).

Except for displacement ventilation, other ventilation types combined with radiant floor systems have received less research attention [[31\]](#page-19-0). Mixing ventilation is commonly used in traditional air conditioning and maintains lower vertical air temperature differences in the room [[89,97](#page-20-0)]. The better ventilation results can be accomplished when the supply air temperature equals or is lower than the room temperature for mixing ventilation [[98\]](#page-20-0). Mixing ventilation mainly dilutes the air to maintain indoor air quality, but it is not efficient in removing pollutants from dead spaces in the room under cooling conditions. The combination of mixing ventilation with a radiant floor system could elevate the risk of airborne pollutant transmission due to reduced ventilation volume. In fact, mixed ventilation is more typical in radiant floor heating systems than in RFCS [\[31](#page-19-0)].

Personalized ventilation systems supply fresh air directly to the personnel breathing zone and are linked to indoor facilities [\[99](#page-20-0)]. Due to the need for duct design in personalized ventilation systems, which can affect the layout and aesthetics of the room, ductless personalized ventilation systems have been developed. Ductless personalized ventilation systems intake fresh air distributed in the upper zone near the floor and transports it to the breathing area of each user [\[100\]](#page-20-0). The application of ductless personalized ventilation systems requires the cleanliness of the air in the upper zone near the floor, often used with displacement ventilation. Liu et al. [[101](#page-20-0)] reported that combining ductless personalized ventilation with radiant floor and displacement ventilation systems can create an improved local thermal environment and pollutant removal effect. The ductless personalized ventilation reduces the vertical air temperature difference between 1.1 m level and 0.1 m level by 1.79 \degree C. It is recommended a flow rate of 5 L/s and an air intake height of 0.1 m for ductless personalized ventilation.

Like personalized ventilation systems, underfloor ventilation systems also require available underfloor space. The heat transfer performance of the radiant floor is affected by wind speed and underfloor air supply temperature [\[102\]](#page-20-0). Fernández Hernández et al. [[103](#page-20-0)] have developed a novel composite floor air distribution system (refer to Fig. 9) and found

that the interaction between underfloor ventilation system and the radiant floor system is not immediately evident. The composite system not only enhances chiller efficiency and augments the cooling capacity of the radiant floor system but also prevents condensation on the radiant surface. The underfloor ventilation modality is suited for public buildings or residential buildings where the diffusion area is free of pollutants.

2.3. Combination with radiant terminals

The cooling capacity of a radiant floor system is the key point of its application, the energy-saving potential of a radiant system increases as it takes on a greater indoor load. Radiant floor system can be used with other radiant terminals within a building to enhance cooling capacity [[72\]](#page-19-0). There are three main methods of combining radiant terminals ([Fig.](#page-8-0) 10): radiant floor + radiant ceiling [[104,105\]](#page-20-0), radiant floor + radiant wall $[6]$, and radiant floor + radiant wall + radiant ceiling $[51]$ $[51]$. Due to the dominant role of heat transfer in the active chilled beam systems, they are not discussed in this chapter due to the dominant role of convective heat transfer [[3](#page-18-0)]. The most common combination is the use of radiant floor and radiant ceiling which is prevalent in thermally activated buildings. These buildings use non-insulated pipes embedded in the floor slab to evenly distribute heat upwards and downwards, being referred to as radiant floor/ceiling cooling system [[105](#page-20-0)]. The asymmetric thermal radiation can be reduced by a symmetrical configuration of the radiation surface [\[106\]](#page-20-0). Furthermore, the combination of radiant walls and radiant floor can provide uniform thermal comfort [[107](#page-20-0)]. Carli et al. [[6](#page-18-0)] conducted a study on an office building in Austria that used a combination of radiant walls and radiant floor and found that the operating temperature of an office was below 26 ◦C during the cooling period. Finally, the integration of phase change materials on different radiant surfaces can create cool surface effects and provide a new avenue for improving indoor thermal comfort [\[51](#page-19-0)].

To improve the efficiency of radiant cooling systems while ensuring indoor thermal comfort, integrating various radiant surfaces is effective. However, there are still certain issues that require attention. The primary area of concern is the cooling and heating capabilities of the radiant terminals and distribution of the indoor load. Tian et al. [\[61](#page-19-0)] conducted experiments in Canadian climate conditions to analyze the thermal comfort and thermal environment of radiant floor/ceiling cooling system. The results indicate that 67 % of the subjects felt overcooling in summer due to inappropriate system design. Another issue to be considered is the response time of the system, which can vary significantly among different types of radiant systems, posing challenges in controlling the system [[18\]](#page-18-0). Finally, condensation risk on radiation surfaces can also be problematic. To combat this issue,

Fig. 9. Radiant cooling floor integrated with an underfloor air distribution system [\[103](#page-20-0)].

Fig. 10. Composite diagram of radiant floor and other radiant terminals: (a) Radiant floor + radiant ceiling, (b) Radiant floor + radiant wall, and (c) Radiant floor + radiant wall $+$ radiant ceiling.

anti-condensation measures should be comprehensively designed and implemented across each surface [[108](#page-20-0)].

3. Control strategies

3.1. Current research status

The control issues of air conditioning include nonlinear dynamic behavior of the system, variable and uncertain disturbances, timevarying set points, passenger-centered ventilation nature, integration of modern technologies, and energy efficiency [\[109,110\]](#page-20-0). Radiant floor systems require more complex control methods due to the small temperature difference between the indoor temperature set point and the cold source, as well as the high thermal mass of radiant surfaces.

The traditional control strategies are effective [\[111\]](#page-20-0), but advanced control strategies have been shown the potential to significantly reduce energy consumption [[81\]](#page-20-0). The primary aim of control strategy is to achieve better thermal comfort and energy performance by regulating supply water temperature, supply water flow rate, supply air temperature, and supply air flow rate [\[41](#page-19-0)]. Indoor control parameters include indoor air temperature, operative temperature, predicted mean vote (PMV), and indoor dew point temperature. Traditional control strategies make control decisions based on identifying changes in indoor parameters, which may pose a risk of indoor overheating [\[88](#page-20-0)]. To reduce control delays, control actions are predicted in advance based on disturbances. Nonetheless, the mutual influence between temperature and humidity control can lead to increased control error $[27,112]$ $[27,112]$ $[27,112]$. As a result, various control strategies have been studied for RFCS to ensure both thermal comfort and cost-effectiveness. [Table](#page-9-0) 2 provides an overview of the current state of research on control strategies for the RFCS. About two-thirds of the research have been conducted via simulation, as it makes simulation methods more convenient and comprehensive for conducting systematic multi-factor analysis. While experimental evaluations have demonstrated the efficacy of complex control strategies, comparatively little research has been conducted on the practical

operation and management of these strategies.

3.2. Control method

According to the control classification of the heating, ventilation, airconditioning and cooling system, the control strategy of the RFCS can also be divided into four types [\[109\]](#page-20-0): conventional control, rule-based control, hybrid, soft-computing control, and advanced control.

3.2.1. Conventional control

Conventional control strategies for indoor temperature control include manual control, ON/OFF control, feedback control, and PID control. The traditional ON/OFF control is a basic and uncomplicated control that does not require intricate parameter settings. The frequent switching of system water pumps can be prevented by applying a dead zone control at the set point. At the same time, energy consumption will be reduced [[21,](#page-18-0)[105](#page-20-0),[115](#page-20-0)]. However, the system responds to indoor environments through on/off control, which can lead to temperature lag under high thermal inertia and variable climate conditions [[65\]](#page-19-0). Many current buildings have added digital control, and PID control is mainly based on feedback closed loop, it is mainly used in modern buildings with variable frequency motors [[109](#page-20-0)]. Employing PID control, floor surface temperature can be precisely regulated by controlling the indoor air temperature with an accuracy of \pm 0.05 °C [\[128\]](#page-21-0). Conventional control strategies are commonly adopted in real building environments due to their cost-effectiveness and simplicity. However, RFCSs exhibit nonlinearity and slow response characteristics, and improper selection of the controller parameters could result in its unstable output [[129](#page-21-0)]. Engineers need to invest significant time and knowledge in tuning and calibrating the PID parameters (proportional, integral, and derivative). Reliable and sensitive algorithms are needed to assist in selecting control parameters for adaptive changes in system parameters.

3.2.2. Rule-based control

Rule-based control is an interesting optimization method, including

Table 2

A summary of the literature on the control strategies applied to the RFCS.

(*continued on next page*)

Table 2 (*continued*)

Note: TABS: thermo-active building system, VS: ventilation system, RF: radiant floor system, RFC: radiant floor/ceiling system, AHU: air handling unit, DOAS: dedicated outdoor air system, FC: fan coil.

gain scheduling, state-space multivariable, transfer function controllers, and nonlinear controllers. It operates on a series of predefined rules, building models through "if-then-else" statements [\[130\]](#page-21-0).

Gain scheduling uses a set of linear controllers to regulate nonlinear system [[131,132\]](#page-21-0). The scheduling variable of the gain scheduling can be the outdoor temperature, while the controlled variable can be the radiant layer temperature [\[32](#page-19-0)]. The technique can leverage previous operating data to optimize control variables and parameters and handle imprecise and uncertain inputs to produce signal responses. Therefore, it can provide a better solution for different radiant floor systems with different thermal masses [\[133\]](#page-21-0).

State-space variable and transfer function approaches can be used to model radiant floor systems mathematically [\[18](#page-18-0)[,134\]](#page-21-0). These methods enable the calculation of cooling loads or describe the energy interaction between the radiant floor and the room. Radiative and convective heat transfer coefficients can be determined through empirical values or experimentation. The delay time between the heat input of the radiant floor system and the regional air temperature can be determined based on the transfer function between the cold floor and the regional air temperature. This allows for a reduction in peak power and energy consumption of the system [\[135\]](#page-21-0). Rule-based control serves as the foundation for complex control, and it requires assumptions and simplifications based on the experience of designers when setting up the algorithm model. However, these assumptions and simplifications may limit the adaptability of the model.

3.2.3. Hybrid, soft-computing control

Hybrid, soft-computing control methods, such as artificial intelligence, fuzzy logic, genetic algorithm, nonlinear hybrid, and data-driven control, do not require complex learning procedures and can optimize control parameters by incorporating relevant empirical knowledge. Su et al. [\[136\]](#page-21-0) used a genetic algorithm reverse neural network to optimize the anti-condensation control of the RFCS. The weights and thresholds of the reverse neural network were optimized using genetic algorithm to accurately forecast the pre-dehumidification time and energy consumption of the office. This allowed the designer to select the best control strategies. However, the results obtained through soft computing methods cannot determine whether they are the optimal solution [\[137](#page-21-0), [138](#page-21-0)].

Various methods are constantly reorganized and improved, making "soft computing" more and more refined [[139](#page-21-0)]. Hybrid and soft computing methods have also been applied to analyze the dynamic performance of radiant floor systems, aiming to improve the efficiency

of mining control data. Qin et al. [\[140\]](#page-21-0) proposed a data-driven fuzzy logic model. The model utilized outdoor air temperature, outdoor relative humidity, and the average temperature of supply and return water as input, and floor surface temperature, indoor air temperature, and indoor air humidity as output. The method fuzzed the coupling variables of indoor air temperature and floor surface temperature. It was discovered that the hysteresis between floor surface temperature and indoor air temperature was insignificant.

Hybrid, soft-computing control is inherently nonlinear and does not rely on precise computational models or extensive calculations. Compared to traditional methods, it offers advantages in addressing issues of uncertainty, imprecision, and optimization.

3.2.4. Advanced control

The use of RFCS to control indoor environments is complicated due to various indoor and outdoor factors. The thermal inertia of building can affect the operating performance of the system. Advanced control strategies can address the complexity and limitations involved in balancing building energy consumption and indoor thermal comfort. By implementing advanced control in near-zero energy buildings with RFCS, higher energy efficiency and improved comfort can be achieved compared to original control strategy of the building [\[141\]](#page-21-0).

Advanced control mainly includes model predictive control and adaptive control and requires comprehensive data and models challenging data and models. Its optimization targets are usually indoor thermal comfort and energy when applied to RFCS controls. Previous studies have investigated the application of advanced control in air conditioning systems [\[26](#page-19-0)[,133,142](#page-21-0)]. The model predictive control can optimize the operating state of system in a future period by predicting disturbance. Weather forecasting and internal load forecasting provide effective basis for the execution of the method [[15](#page-18-0),[143](#page-21-0)]. Adaptive control can perform TABS control for specific regions based on multiple linear regression and continue to learn and improve, allowing it to address indoor disturbances in a timely fashion [[13\]](#page-18-0). The stability of advanced control can be threatened by several factors, and improvements in systems and models can mitigate these issues [\[109](#page-20-0)]. Despite the rapid development of hardware technology, model predictive control for RFCSs requires longer computational time for prediction. The complex design and high cost brought using advanced control in actual buildings still need to be considered.

3.3. Anti-condensation control

The building cooling load consists of both latent heat load and sensible heat load. Radiant floor system can only handle the sensible load, and cannot bear the indoor latent heat load. Excess humidity will cause discomfort for occupants, condensation on the radiant surface, and reduction of performance of radiant floor heat transfer. The cause of condensation on the radiant floor is when the surface temperature drops below the indoor dew point temperature. To address this issue and improve the cooling capacity of the system, it is important to lower the temperature of the radiant surface while lowering the indoor dew point [[41](#page-19-0)[,144\]](#page-21-0). The high thermal mass RFCS has been evaluated in hot and humid climates and found to be effective in multi-zone residential buildings [[145](#page-21-0)].

The minimum recommended radiant floor temperature in the standard is 19 \degree C [\[14](#page-18-0)]. However, in actual buildings, the furniture will cause uneven temperature on the floor surface and make it difficult to accurately control the temperature of the floor surface. To prevent condensation, an anti-condensation method is used by maintaining the supply water temperature $1-2$ °C higher than the indoor dew point temperature or turning off the system when the floor temperature drops below the indoor dew point temperature [[146](#page-21-0),[147\]](#page-21-0). Based on the experimental study, the type of the radiation panel structure can influence the condensation limit, resulting in the condensation temperature difference between the dew point temperature and floor temperature reached 3.2 \sim 6.5 °C. The waste of anti-condensation energy consumption can be reduced when anti-condensation applied in system based on the sub-cooling degree [[148](#page-21-0)]. Compared with flow rate control, water temperature control is more effective in responding to radiant floor surface anti-condensation [[149](#page-21-0)]. In addition to regulating radiant floor heating, the ventilation system also plays a crucial role [[23,25](#page-18-0)]. It can be controlled by variable air volume according to different zones, schedules, or indoor humidity. Lim et al. [[147](#page-21-0)] found that the combination of outdoor temperature feedback control and dehumidification control can reduce the operating time of the dehumidification system by half.

During the startup phase of the radiant floor system, there is also a high risk of condensation on the radiant surface. Pre-dehumidification control is used to solve the condensation problem during opening [[136](#page-21-0),[150](#page-21-0)]. Pre-dehumidification refers to opening the ventilation system for a period before the start of the radiant floor system to ensure that the indoor environment meets the necessary standards. The control of pre-dehumidification time can avoid wasting unnecessary initial energy consumption, which can be calculated or predicted based on the thermal inertia parameters of the radiant floor [[150](#page-21-0)].

The pre-dehumidification time is a critical factor in preventing condensation from forming on the radiant floor surface. Regrettably, only limited studies have analyzed this important aspect.

3.4. Evaluation indices for the control strategies

The comparison between different strategies can be divided into short-term and long-term evaluations. The short-term evaluation focuses on hourly comparisons of thermal comfort and energy consumption of different control strategies in a typical period. This evaluation will visually display indoor and system parameters using graphical representations [[121](#page-20-0)]. It offers a detailed analysis of strategy operation, including parameter trends and system conditions. On the other hand, the long-term evaluation aims to assess the overall stability and performance of strategies throughout their entire cycle. It calculates the distribution hours of different parameter ranges, the average value of parameters, and the sum of parameters in the cycle [[14,](#page-18-0)[105](#page-20-0),[127](#page-21-0)]. Thermal comfort analysis mostly focuses on calculating the hours of the parameter distribution, expressed through the proportion of the distribution time, since mean values cannot represent extreme value parameters. The evaluation parameters of different strategies fall into three categories: thermal comfort parameters, economic parameters, and

performance parameters.

3.4.1. Thermal comfort

Indoor thermal comfort is a critical criterion for assessing the efficacy of control strategies. It encompasses both general thermal comfort and local thermal discomfort, with thermal comfort range of each evaluation index can be divided into three categories (as displayed in [Table](#page-12-0) 3) [\[14](#page-18-0)].

3.4.1.1. General thermal comfort. The key factors affecting overall thermal comfort can be boiled down to six main ones: metabolic rate, thermal resistance of clothing, air temperature, radiant temperature, air velocity, and indoor air humidity [\[151\]](#page-21-0). Although satisfaction of occupants with the indoor environment can also be used to evaluate thermal comfort, objective evaluative criteria are typically employed when assessing system control strategies. While metabolic rate and thermal resistance of clothing are generally set as fixed values in research due to their relationship with personnel behavior, the evaluation of indoor thermal comfort is primarily based on the last four factors.

Indoor air temperature is often representative of the ambient air temperature of the occupant [[151](#page-21-0)]. Typically, in experimental studies, temperature measuring points are placed at 1.1 m [\[95](#page-20-0)]. The recommended indoor air temperature by the International Organization for Standardization (ISO 7730) is between 24 and 26 ◦C [[14\]](#page-18-0). Compared to traditional convection air conditioning systems, RFCSs can maintain better thermal comfort in buildings [\[117\]](#page-20-0), with indoor air temperatures comfortably ranging between 25.8 and 29.2 ◦C [[152](#page-21-0)]. The radiant floor system exchanges heat with the indoor heat source through both convective and radiant heat transfers and lower mean radiation temperature can increase the tolerance of indoor air temperature. It is essential to consider the indoor mean radiant temperature. However, evaluating thermal comfort can be achieved more comprehensively using operative temperature since it represents the impact of both air temperature and mean radiant temperature within the indoor environment. The equation for calculating indoor operative temperature is shown in Equation (1) [[151](#page-21-0)], and [Table](#page-12-0) 3 displays the recommended operative temperature ranges based on clothing insulation and metabolic rate of occupants.

$$
t_{o} = At_a + (1-A)\overline{t_r} \tag{1}
$$

where t_0 is the operative temperature; t_a is the average air temperature; $\overline{t_r}$ is the mean radiant temperature; *A* is a coefficient as a function of the average air speed (When the average air speed *<*0.2 m/s, *A* is 0.5; when the average air speed is in the range of 0.2–0.6 m/s, *A* is 0.6; when the average air quality is in the range of 0.6–1 m/s, *A* is 0.7.)

Radiant floor systems can follow the design values of conventional all-air systems for relative humidity and wind speed [\[153\]](#page-21-0). Under the standard indoor temperature of 26 ◦C in summer, the indoor relative humidity range needs to be controlled at 40–60 % [\[14](#page-18-0)]. The thermal discomfort caused by higher indoor relative humidity can be compensated by lower mean radiant temperature [[152](#page-21-0)], or integrated with dehumidification equipment to overcome. The indoor relative humidity design values for rooms that use dehumidification are shown in [Table](#page-12-0) 3 [[154](#page-21-0)]. Since the system often works in conjunction with ventilation systems, Indoor air velocity need to be considered. To increase energy efficiency and reduce consumption, ventilation volume is based on indoor carbon dioxide concentration and minimum air change ratio in the room [\[71](#page-19-0)[,155\]](#page-21-0). This ensures indoor fresh air demand and keeps the indoor air velocity below the limit of 0.24 m/s. Although some tropical countries have local standards for minimum indoor wind speed [[35\]](#page-19-0), it is rarely used as an evaluation index for control strategies.

Thermal comfort is an essential factor in evaluating indoor environments, and predicted mean vote/predicted percentage dissatisfied (PMV/PPD) is the most commonly used framework [[27\]](#page-19-0), mainly used to evaluate instantaneous thermal comfort indoors. For indoor parameters involving a certain time period or operating cycle, it is necessary to **Table 3**

Range of different thermal comfort parameters [[14,](#page-18-0)[154\]](#page-21-0).

Note: ^a The criteria for Vertical air temperature difference is 1.1 and 0.1 m above floor.

 b The criteria for operative temperature and maximum mean air velocity are based on typical levels of activity, for clothing of 0.5 clo in summer.</sup>

quantify the number of hours beyond the comfort zone based on measurement or simulation results. The equations for calculating the number of hours beyond the PMV comfort zone and the adaptive model comfort zone is as follows [\[151\]](#page-21-0):

$$
EH_{\rm p} = \sum H_{\rm disc} \tag{2}
$$

where EH_p is the exceedance hours calculated for the PMV comfort zone; H_{disc} is a discomfort hour ($H_{\text{disc}} = 1$ if $|PMV| - 0.5 > 0$ and 0 otherwise).

$$
EH_{t} = \sum (H_{\geq \text{upper}} + H_{\leq \text{lower}})
$$
\n(3)

where EH_t is the exceedance hours calculated for the adaptive model comfort zone; H_{super} and $H_{\text{ slower}}$ is are discomfort hours outside the ϵ comfort zone boundaries $[t_{lower}, t_{upper}]$, $(H_{>\text{upper}} = 1$ if $t_{\text{op}} > t_{\text{upper}}$ and 0 otherwise; H_{5} lower = 1 if $t_{\text{op}} < t_{\text{lower}}$ and 0 otherwise).

However, recent research found that PMV tends to overestimate the thermal sensation of occupants [[156](#page-21-0)], especially when evaluating different system configurations [[96\]](#page-20-0). This is mainly affected by uneven thermal conditions, and the difficulty in quantifying the view factor between the human body and the surrounding environment results in simplified calculations for mean radiant temperature [[157](#page-21-0)], which in turn affects the values of operative temperature and PMV.

3.4.1.2. Local thermal discomfort. The evaluation of local discomfort includes vertical air temperature difference, warm/cold floor, and radiant temperature asymmetry [\[14](#page-18-0)]. The RFCS can create a vertical air temperature difference [[94\]](#page-20-0). The vertical temperature gradient is influenced by the form of ventilation. It should be within 3 K from the ankle level to the head level $[158]$ $[158]$ $[158]$. The range of air temperature difference from the ankle level to the head level should also be limited. If the vertical air temperature difference is below 8° C, Equation (4) can be used to determine the local discomfort (PD_1) caused by it $[14]$ $[14]$.

$$
PD_1 = \frac{100}{1 - \exp(5.76 - 0.856 \Delta t_{a,v})}
$$
(4)

where PD_1 is the local discomfort caused by vertical air temperature difference; $\Delta t_{a\nu}$ is the vertical are temperature difference.

The radiant floor system focus on the lower body and can achieve similar levels of thermal comfort at a higher surface temperatures [[159](#page-21-0)]. Nevertheless, the human hands and feet are more sensitive to thermal comfort, and cold floors can easily cause local discomfort and coldness [[160](#page-21-0)]. The investigation is required to comprehensively understand the subjective thermal sensation experienced by individuals on cold floor. The recommended minimum floor temperature is 19 ◦C in areas where sedentary or standing individuals wearing standard footwear are present [[158](#page-21-0)]. The standardized threshold for localized thermal comfort can vary significantly depending on temporal conditions and requires appropriate adjustments based on the floor surface material and occupant behavior [\[95](#page-20-0)]. Equation [\(5\)](#page-13-0) demonstrates the effect of a cold floor (PD_2) on local thermal discomfort $[14]$ $[14]$ (Equation (5) does not consider the long-term operation of electrically heated floors or situations involving barefoot contact by occupants).

$$
PD_2 = 100-94 \cdot \exp(-1.387 + 0.118t_f - 0.0025t_f^2)
$$
 (5)

where PD_2 is the local discomfort caused by cold floor; t_f is the temperature of the floor surface.

There are temperature variations in the radiation between different surfaces within indoor environments, thus an evaluation of the indoor temperature asymmetry is needed within the system. The room with RFCS can easily result in vertical temperature differences in the radiation. It can lead to discomfort when exposed to for extended periods [[161](#page-21-0)]. Among the three local thermal discomfort indicators, the impact of radiation temperature asymmetry on discomfort is least described in relation to cold floors. Radiation temperature asymmetry can be used to quantify the effects of indoor uneven radiation. ASHRAE 55–2017 [[151](#page-21-0)] set the maximum allowable temperature difference of 5 ◦C. The non-symmetric radiation effect cannot be balanced out between different parts of the human body, while outdoor conditions and the building envelope can impact the exchange of thermal radiation with the human body [\[159\]](#page-21-0). Occupiers located near windows may be subjected to non-symmetric radiation from multiple directions. Currently, there is limited research that considers indoor radiation asymmetry as an indicator when evaluating system control strategies.

3.4.2. Economy

Economic evaluation is a crucial factor in determining the feasibility of control strategies. Evaluation of economy considers both the initial investment cost and the operating cost of the system. However, it can be challenging to quantify these costs accurately due to variations in regional conditions. There are no specific normative constraints on evaluation scope, and the current evaluation criteria are based on the principle of "less is better." As a result, costs are often minimized when incorporated as an expected goal in control strategies [[13\]](#page-18-0).

The evaluation of system operating costs for different control strategies mainly involves comparing system energy consumption or electricity costs. Energy consumption can be broken down into partial energy consumption and overall energy consumption, with refrigeration equipment and transportation energy making up the overall energy consumed. Comparisons for the entire cooling season are usually made for RFCS, and experimental testing may reduce the comparison time. Some regulations [\[155\]](#page-21-0) stipulate an upper limit for the average energy consumption index of newly built buildings. On the other hand, the difference of electricity costs is mainly affected by the operating time of the system, the thermal inertia of the system, and the peak valley electricity price [[162](#page-21-0)]. While initial investment is minimally affected by the control strategy, different methods do require different configurations of system components and control accuracy, leading to cost differences [[81,122\]](#page-20-0). The economic gap between two compared radiant floor cooling systems can be measured by the payback period, which refers to the number of years needed for reduced operating costs to repay the high initial investment cost of the system. The evaluation of payback period is of great significance for high-cost and low-energy consumption RFCSs, such as systems utilizing renewable energy or new radiant floor structures.

3.4.3. Performance of systems

To optimize system performance and efficiency, it is important to consider internal and external adjustments. Exergy is the part of energy that can be maximally converted into work when the system undergoes a

reversible transformation from any state to equilibrium with the reference state. It emphasizes the efficient utilization of energy and is an important concept in energy analysis. Like economic evaluation, there is no specific scope or limitation for exergy analysis. However, it is an effective tool used to assess operating performance and potential of a system for optimization under varying conditions [\[163\]](#page-21-0). Exergy analysis is based on combining the first law of thermodynamics with the second law of thermodynamics [\[164\]](#page-21-0). This analysis can determine the impact of operating temperatures and energy and flow rates within the system [[165](#page-21-0)]. Moreover, exergy analysis can also be utilized as a method for system evaluation. To analyze the system, an equilibrium equation must be established in advance, as demonstrated in Equation [\(4\)](#page-12-0) [[69,](#page-19-0)[163](#page-21-0)].

Exergy input $+$ Exergy consumed $=$ Exergy stored $+$ Exergy output (4)

Exergy analysis often overlooks the causes of exergy loss and the utilization of internal resources. The commonly used evaluation indicators include exergy loss and exergy efficiency. Exergy loss is a decrease in energy quality in a system, mainly occurring in cold source and transportation systems. Exergy analysis can evaluate exergy loss and efficiency. Exergy efficiency refers to the percentage of exergy used effectively in the input system. The closer the exergy efficiency approaches 1, the better the thermodynamic perfection of the system. The calculation equation for efficiency is shown in Equation (5) [[69\]](#page-19-0).

$$
\eta_{ex} = 1 - \frac{\text{Exergy Loss}}{\text{Exergy input}} \tag{5}
$$

The RFCS is efficient as a low-exergy system composed of multiple subsystems [[69\]](#page-19-0). Dealing with small temperature differences, it is possible to minimize the exergetic value of the heat flux in relation to its energetic value [\[166,167\]](#page-21-0). However, conducting thermal analysis can be complex due to numerous design parameters. To improve accuracy and efficiency, optimizing the calculation process is necessary.

4. Application

4.1. Case studies

The use of radiant floor for heating is widespread, it is also effective for building heating during winter [[149\]](#page-21-0). Nevertheless, there are relatively few architectural reports on radiant floor heating and cooling system. The cases where they have been implemented in practical buildings are summarized in [Table](#page-14-0) 4. It is evident that buildings equipped with radiant floor heating and cooling systems exhibit high energy efficiency and provide significant economic benefits. This can be attributed to the use of natural cold sources like cooling towers or direct-ground cold source. Theoretically speaking, the natural cold source may be impacted by the environment, leading to unstable water supply temperatures, but performance results during testing have been consistently satisfactory. Excessive use of natural cold sources can be avoided by optimizing the utilization time of natural cold source [[168](#page-21-0)]. The investigation of indoor thermal comfort is divided into measurement and questionnaire surveys. Although the radiation surface temperature is uneven due to pipeline laying and other factors [[122](#page-20-0)], little research has explored the temperature distribution of radiant surface and asymmetric radiation. The occupants and test results generally meet the requirements for thermal comfort, barring special human interference. However, overcooling or overheating may still occur in certain periods [\[169,170](#page-21-0)]. In addition, the dehumidification control of the system can be affected by the behavior of residents [\[145](#page-21-0)]. In conclusion, the application results of the radiant floor heating and cooling system have achieved good application, especially in energy-saving buildings.

4.2. Climate types

Climate is an important factor in the application of RFCS, which can

be classified into four types based on temperature and humidity: high temperature and high humidity, high temperature and low humidity, and low temperature and high humidity [[84\]](#page-20-0). RFCS is subjects to climatic disturbances mainly from solar radiation, infiltration, and ventilation [[176](#page-22-0)]. RFCS is used in conjunction with ambient energy to condition the building, the climate not only impacts the thermal comfort evaluation of the building's interior but also its energy efficiency [\[13](#page-18-0)[,78](#page-20-0), [177](#page-22-0)].

In low humidity areas, RFCS can operate independently in the indoor environment based on the control strategy used. But in high humidity areas, as mentioned in section [2.3](#page-7-0), radiant floor system needs to be integrated with ventilation systems to overcome climate effects [\[46](#page-19-0),[127](#page-21-0)]. The use of RFCS has grown quickly in high-humidity areas, and it promotes energy efficiency in high humidity environments [\[11](#page-18-0)]. Li et al. [[177](#page-22-0)] found that RFCS is suitable for all climate zones in the United States. In fact, RFCS is more energy efficient in environments with lower humidity than in environments with higher humidity due to the high energy consumption of cooling and dehumidification systems [[23,](#page-18-0)[84](#page-20-0)]. The pre-cooling strategy can be used to reduce the peak cooling load and increase the cooling capacity in high temperature climate. Chandrashekar and Kumar [\[67](#page-19-0)] performed pre-cooling on a RFCS in a university building and found that the pre-cooling effect shifted the peak indoor temperature by 2 h and reduced energy consumption. RFCS demonstrates impressive self-regulating capabilities in high solar radiation environments. The radiant floor has a cooling emission rate of 39–50 $W/m²$ when facing a mixture of cooling loads (convection and radiant), while the radiant floor has a cooling emission rate of 150–226 $W/m²$ when only receiving short wave radiation (solar energy) [[178](#page-22-0)]. However, the intermittent operation of the system would be significantly influenced by the intensity of solar radiation [\[179\]](#page-22-0). The application of RFCS in transitional seasons is easy to cause simultaneous cooling and heating, which leads to energy waste [\[20](#page-18-0)].

With the influence of human activities on the environment, the climate environment in the same area is also constantly changing. To promote the application of RFCS in various regions of the world, the recommendations for the effective implementation of RFCS in different climates are summarized [\(Fig.](#page-15-0) 11). Normally, the recommendations are affected by additional limitations, for example, natural night ventilation needs to consider rainy days and air pollution. Further research is needed on the optimal operating parameters of RFCS under different climate types to avoid both condensation on radiant surfaces and energy inefficacy [[127](#page-21-0)]. Some traditional control strategies have been combined with climatic conditions, adjusting water supply parameters of RFCS according to the outdoor conditions [\[11,24](#page-18-0),[180](#page-22-0)]. Advanced control strategies [[15,](#page-18-0)[112\]](#page-20-0) also use weather forecasting to predict and correct the next operation of the system, thus maintaining better thermal comfort in the room.

4.3. Building types

RFCS can be used in many types of buildings, such as office buildings [[11](#page-18-0)[,181\]](#page-22-0), residential buildings [[25,](#page-18-0)[145\]](#page-21-0), hospitals [\[177\]](#page-22-0), educational buildings [\[19](#page-18-0)], museums [\[182\]](#page-22-0), airports [[30](#page-19-0),[183](#page-22-0)], etc. The physical characteristics of the building will affect the system type and control strategy selection of the RFCS and indoor thermal comfort [[176](#page-22-0)].

Most of the current research on RFCS is focused on commercial buildings [\[184\]](#page-22-0). These buildings have a small shape factor that makes them less susceptible to outdoor climate. Moreover, the working hours occupancy pattern of some commercial buildings is relatively uniform. Due to commercial buildings tendency to maintain positive pressure, the changes in indoor humidity load caused by infiltration can be ignored [[13\]](#page-18-0). Although indoor positive pressure is beneficial for reducing the pollution of outdoor air, it can cause moisture to penetrate the building envelope. So, in high outdoor air quality or dry climates, balanced ventilation or slight negative pressure will be preferred indoors. Talami and Jakubiec [[17\]](#page-18-0) performed a sensitivity analysis of 13 design

Table 4

Application cases of radiant floor cooling and heating system.

Reference	Year	Location	Climates	Building type	System	System cold/heat source	Monitoring period	Building diagram
[171]	2005	Hamburg, Germany	Temperate marine climate	Office building	$VS +$ \mathbf{RF}	CHP-plant, the condensing boiler, direct-ground cold source	July 2002-Aug. 2002	
$[19]$	2007	Biberach, Germany	Temperate marine climate	Vocational training school	$VS +$ $\rm RF$	Groundwater, heat pump, boiler	2005	
$[19]$	2007	Aachen, Germany	Temperate marine climate	Office building	$VS +$ \mathbf{RF}	Borehole heat exchangers, heat pump	2005	
[169]	2009	Calgary, Canada	Temperate continental climate	the ICT Building at the University of Calgary	$VAV +$ RFC	Cooling coils, the campus water network	2004, 2006	
[168]	2010	Germany	Temperate marine climate	House	$VS +$ RFC	Rainwater storage tank, heat pump, solar collector	2006-2007	COOLING VENTILATION HEATING
[83]	2015	Xian, China	Moist continental warm temperate monsoon climate	Airport	$AHU +$ \mathbf{RF}	Cooling tower, water chiller, heat exchange station		
[172]	2016	Ancona, Italy	Subtropical humid climate	the Leaf House	$AHU +$ $\rm RF$	Geothermal heat pump, ground coupled heat exchanger solar collector, boiler	2010	
[173]	2016	Denmark	Temperate marine climate	House	$AHU +$ $\rm RF$	Heat pump	Sep. 26, 2013-10 Oct. 2014	
$[174]$	2017	Colorado, United States	Temperate continental climate	Educational building	$VS +$ \mathbf{RF}	Cooling tower, natural gas condensing boiler system	2016-2017	

(*continued on next page*)

A,

 $\frac{1}{2}$, $\frac{1}{2}$

Fig. 11. Summary of suggestions for the application of RFCS in different climate types.

parameters for office buildings with RFCS installed, found that solar heat gain coefficient and area ratio of window to wall played a key role in thermal comfort, operation, and energy consumption of RFCS. It was determined that buildings with square or compact rectangular shapes achieved better energy performance. The control of a system is typically based on the difference in heat loss or gain and the heat or cooling required for the typical room [\[41](#page-19-0)]. The commercial building area is relatively large, and the buildings will be divided into areas based on the north-south direction to ensure the accuracy of system control [\[41](#page-19-0)]. In different commercial building types, the average energy saving of RFCS varies between 6.84 % and 48.0 % [\[177\]](#page-22-0).

The behavior of occupants in residential buildings is a crucial factor in the application of RFCS. Random occupant behavior significantly affects the cooling demand and thermal comfort of the RFCS, and a good control strategy needs to adapt to various occupant behaviors [\[126](#page-21-0), [185](#page-22-0)]. The research on the control of RFCS in residential buildings focuses more on conventional control strategies (as shown in [Table](#page-9-0) 2). Unlike in office buildings, the control strategies of RFCS in residential buildings are more reliant on external climate conditions and conventional strategies [[41\]](#page-19-0). Furthermore, the use of lighter-weight radiant floor in residential buildings can also reduce the response time of the system and provide better thermal comfort for the interior. Andersen et al. [\[71](#page-19-0)] studied the operation of the RFCS in residential buildings and discovered that the RFCS generates higher peak temperatures than the

all-air cooling system. Due to the high insulation standards of buildings and the low energy efficiency of the system during nighttime, apartments or single-family residences are usually controlled as one control area [[41](#page-19-0)].

The building envelope structure connects the indoor climate environment with the external environment, and its insulation performance can have a significant impact on building energy consumption [[186](#page-22-0)]. For building envelope structures using RFCS, their thermal storage capacity, permeability, and internal surface heat transfer need to be considered $[23,82]$ $[23,82]$ $[23,82]$ $[23,82]$. The thermal resistance of the envelope can significantly impact system operation [[187](#page-22-0)]. Radiant floor systems cool the building envelope, which can lead to long response times and wasted energy when the envelope has considerable heat storage capacity. In fact, RFCS has shown positive results when applied in high-rise buildings [[19\]](#page-18-0). Large surface areas of transparent materials are utilized in buildings, causing excessive internal surface heat and transmission of solar radiation [[107](#page-20-0)]. Large space buildings can vertically divide space, effectively cooling only the occupied space. The radiant floor will be maximized cooling capacity under solar radiation, which will also increase the temperature difference between the supply and return water [[83\]](#page-20-0). It is recommended to combine natural ventilation, natural cold source, and sunshade design for optimization. If the high-quality interconnection and linkage between the internal functions of buildings and the radiant floor cooling systems are achieved, it is expected to bring

more opportunities for the radiant floor cooling systems.

5. Discussion

5.1. Floor structure and ventilation

Although several conventional floor structures have been proposed in standards, there is still a lack of detailed research and data references on the characteristics of new composite materials. Radiant floors treat the sensible heat load indoors through both radiant and convective heat transfer, and the thermal performance of multi-layer floors would be significantly impacted by the pipe spacing and surface material. The piping materials (PE-X, PE-RT, PB, and PP, etc.) and surface materials of radiant floor heating systems of radiant floor systems, should possess excellent thermal conductivity. In contrast, insulation materials need to have good insulating properties to reduce downward heat loss. Research on the composition materials of radiant floor systems, particularly in studying the thermal conductivity under various operating conditions, is primarily conducted through experimental methods. It should be noted that the surface temperature of the floor is non-uniform, and it is influenced by solar radiation and furniture arrangement. This is not only hindering anti-condensation but also hampering cooling. The inclusion of a high thermal conductivity matrix in the floor can effectively improve the issue [\[52](#page-19-0)]. Accurately estimating the thermal performance of radiant floor systems is beneficial for optimizing floor structures. However, there is limited research on how integrating different types of radiant floor systems into a system affects long-term energy efficiency of the system. In particular, when considering cost variations in innovative designs of radiant floor structures, payback period becomes a crucial criterion in engineering design. It is important to evaluate the extent to which energy savings offset the high costs of radiant floor. Many studies focus on enhancing the responsiveness or cooling capacity of floor [\[29](#page-19-0), [52\]](#page-19-0). For instance, the radiant panel with a short response time may eliminate the need for pre-cooling strategies. Improvement of the construction reliability and anti-condensation performance of radiant floors would hold potential for the application of RFCS.

The ventilation system is the sole bearer of indoor latent load and generally employs cooling dehumidification. Dehumidifiers may also be utilized in situations with high latent load. The capacity of the ventilation system is closely related to the capacity of the radiant system [[188](#page-22-0)]. Numerous studies have documented that the energy efficiency of ventilation systems is lower than that of radiant systems, which motivates minimizing the energy consumption of ventilation. Approximately 24 % of studies set the ventilation air change rate at 0.5–1 $\rm h^{-1}$, 47 % at 1-2 h⁻¹. The remaining 29 % opt to design the air change rate based on the occupants indoors. However, the improper use of the ventilation system may weaken the impact of the radiant floor system on the indoor environment, especially when convective heat gains are dominant [[178](#page-22-0)]. Therefore, it is recommended to improve the floor materials and structure to lower the supply water temperature of the radiant system and increase the capacity of the radiant floor system.

5.2. System operation control

The design of control strategies for RFCS aims to achieve real-time responsiveness to the building interior conditions while reducing energy consumption. Extensive research has been conducted on control strategies, employing simulation and experimentation as the primary research methods. Simulation methods often utilize predefined internal loads and occupant behavior, leading to building load variations that heavily depend on external climate conditions. While abrupt thermal environments provide greater resemblance to actual conditions, there is insufficient research on control strategies in such environments. Occupant behavior and equipment accuracy also affect the implementation of control strategies, resulting in deviations between simulation and experimental results [[75\]](#page-19-0). The importance of precooling strategies in

system control increases as the heat storage capacity of RFCS and the internal loads of buildings increase. Model predictive control shows an 11 %–15 % difference in electricity between RFCS with and without precooling strategies [\[13](#page-18-0)]. Decoupling control of indoor parameters is an interesting approach to mitigate condensation risks while improving operational performance of radiant floor cooling systems. Through literature studies, the adoption of advanced control strategies to replace conventional ones in actual buildings is limited, presenting challenges to their widespread application. The model established in advanced strategies is prone to deviate from the actual building, resulting in control imbalance. Although the model will be validated based on actual building measurement data, changes in equipment and building conditions can compromise the performance of the model. Many studies highlight the need for future optimization of system models. Multi-stage calibration of floor cooling systems can improve model accuracy by 50 % [\[189\]](#page-22-0). Consequently, the development of equipment fault early monitoring and predictive diagnosis [[190](#page-22-0),[191](#page-22-0)] based on RFCS is vital to ensure stable system operation and effective implementation of control strategies.

The initial investment of RFCS can be higher than that of traditional air conditioning systems, and an economic assessment is required to determine its feasibility for application. Exergy analysis provides a more accurate evaluation of the system overall performance. Exergy losses typically occur in transportation and cold source systems, and exergy analysis of RFCS helps explore the system potential for minimizing losses. However, there are relatively few studies on optimizing RFCS operation control based on exergy analysis. Relative comparison can be employed as a method for evaluating control strategies, involving the comparison of multiple cases to a base case or pairwise comparisons [[112](#page-20-0),[123](#page-21-0)]. Many control strategies have been proven strong application potential, but the choice of a base strategy they contrast varies across studies. Therefore, reliable evaluation methods are needed to guide the selection and application of strategies in practical projects.

5.3. Risk of condensation

The primary limitation of RFCS is the occurrence of condensation on the floor surface. Therefore, the surface temperature of the radiant floor is typically maintained above the dew point temperature. When the radiant floor system operates under high loads, measures need to be taken to reduce the dew point temperature to prevent condensation. The improvement of airtightness and insulation performance of the building envelope and incorporation of shading can mitigate outdoor environmental influences [[79\]](#page-20-0). Additionally, an integrated auxiliary system can assist in managing the latent and sensible heat loads inside the space. Different indoor environment of the same building zone needs to be regulated through air systems. In cases of significant latent load, using separate dehumidification and cooling systems is preferable [\[87](#page-20-0)]. Pre-dehumidification is necessary before activating the radiant floor, and due to the thermal inertia of the system, it is still crucial to continue controlling indoor humidity after the radiant system is turned off. Airflow design near the floor and indoor heat source design can also indirectly reduce the condensation risk of floor, but there is significant uncertainty in system operation.

The measures for releasing condensation on radiant floor mainly focus on system regulation, and the design of floor materials is also an important aspect, such as applying a film layer to the radiant surface to isolate the air and prevent condensation ([Fig.](#page-17-0) 12 [\[192\]](#page-22-0)). This film layer ensures the transmission of infrared radiation between the radiant surface and the indoor environment. Consequently, the surface temperature of the film is maintained above the dew point temperature, ensuring satisfactory indoor thermal comfort even when the supply water temperature of the radiant system is low. Passive condensation prevention is highly appealing for the application of RFCS [\[192,193\]](#page-22-0). However, the mechanical performance of materials applied on the surface of radiant floor poses a significant technical challenge. If this challenge is

Fig. 12. Schematic representation of radiation heat transfer for a condensation-free radiant cooling system with double-skin infrared-transparent membranes [[192\]](#page-22-0).

overcome, it will greatly enhance the performance potential of radiant floor systems. Thus, it is recommended to conduct research from various perspectives, such as floor structure and system controls, in order to mitigate the risk of condensation on the radiant surface and simultaneously improve the cooling capacity of the system.

Risk assessment for condensation in RFCS is crucial for the possibility of system application. The condensation rate on the radiant surface of the floor is a basic index to evaluate the condensation on the floor surface $[108]$ $[108]$, although it is seldom utilized in system evaluations. Building simulations provide a means for early assessment of both building dynamics and condensation risks [[183,194\]](#page-22-0). As the maturity of the prediction algorithm, the risk of condensation can be accurately predicted based on models or equations without experiments and simulations [[195](#page-22-0)]. However, despite the potential of this promising approach, its implementation in RFCS has not been reported in the literature.

5.4. Radiant floor heating and cooling system

Radiant floor systems are commonly utilized for heating buildings; however, there are fewer applications that simultaneously integrate both radiant cooling and heating functions. For example, even in residential buildings equipped with radiant floor heating systems, split air conditioning units are still commonly selected for cooling. Compared to building heat load, the peak demand for building cooling load is large and load fluctuates is greater, and the efficiency of centralized cooling sources is low. Variable refrigerant flow system can achieve the same or better performance than RFCS under the same control strategy [\[13](#page-18-0)]. Therefore, it is necessary to explore the building and climate conditions that are suitable for the application of radiant floor cooling and heating systems, and develop efficient composite cooling sources. To achieve optimal cooling and heating effects with radiant floor system, it is essential to consider the heat transfer mechanism and design conditions [[196](#page-22-0)].

With the increase or decrease in the number of buildings, building

renovation and upgrading are gradually being valued [\[197\]](#page-22-0). In the process of construction and renovation, the integration of RFCS into buildings may provide opportunities for further expansion at a lower cost. Building retrofits present challenges beyond installation and maintenance, including complex strategy transitions and limitations of ventilation systems [\[46\]](#page-19-0). Clearly, incorporating thermal storage is encouraged in the design of RFCS, which requires exploration of the power grid and energy storage.

Radiant floor cooling and heating systems can be viable solutions for achieving nearly zero energy buildings [\[75\]](#page-19-0). It is worth noting that increasing the energy storage capacity and reducing the response time of the RFCS are both research hotspots, and the inherent relationship between the two has reference value for the application direction of the system. The coordination between climate and application scenarios should pay careful attention to fully meet energy and thermal comfort requirements.

6. Conclusion

In the context of rapid development in low-carbon energy, the RFCS has emerged as one of the most sustainable air conditioning systems. This review provided a comprehensive overview of the construction of radiant floors, the composition of radiant floor cooling systems, control strategies for systems, and anti-condensation. The review also conducted a thorough review of the application of RFCS to evaluate the balance between innovation and engineering design. Currently, research efforts are focused on the control and optimization, though there are still limitations in practice and innovation. Nonetheless, RFCSs still maintain a vibrant, and investigating the latest developments in various aspects can help in adapting to the energy transition. The main discussions can be summarized as follows:

The control of RFCS is one of the major challenges, which is closely tied to anti-condensation and system design. Consequently, research on RFCS primarily revolves around control strategies. Thermal comfort,

economy, and performance are interconnected and serve as key indicators for selecting suitable control strategies. At present, nonpredictive control strategies remain the dominant control strategies, although significant progress has been made in advanced control strategies. The selection of control strategies has implications for the climate adaptability and building scenarios suitability of the system. Thus, more attention should be paid to the combination of control and application. Condensation risk poses a limitation to RFCS, requiring active dehumidification for effective prevention. The most common RFCS involves integrating the radiant floor system with the ventilation system. However, care must be taken to ensure proper sizing of both systems to avoid compromising the cooling performance of the floor. There is a lack of research on passive condensation methods for investigating floor structures. In contrast, floor structure investigations have focused more on experimental testing. It is crucial to determine the characteristics of new materials and floor structures to provide valuable insights for engineering applications. On the other hand, the application and retrofitting of radiant floor cooling and heating systems are interesting research areas. On the path towards energy conservation and emission reduction, the radiant floor system would stand as a prominent contributor to the effective utilization of clean energy.

The following are proposed recommendations for future research, but not limited to:

- \triangleright The design of radiant floors should prioritize condensation prevention, cost-effectiveness, and high performance to enhance the cooling capacity of such systems.
- \triangleright Anti-condensation measures for RFCS should be better integrated with materials and control to enhance the system adaptability to different climates and building types.
- ➢ Control strategies for RFCS should consider robustness, durability, and system operation fault diagnosis. Advanced control strategies should replace conventional ones in existing buildings.
- ➢ The application of RFCS in buildings should explore options for retrofitting or integrating with radiant floor heating system to expand their scope of application.

CRediT authorship contribution statement

Mengying Cui: Writing – review & editing, Writing – original draft, Methodology, Investigation, Conceptualization. **Baisong Ning:** Writing – review & editing, Supervision, Methodology. **Xiaozhou Wu:** Writing – review & editing, Supervision, Methodology. **Moon Keun Kim:** Writing – review & editing, Supervision. **Bin Yang:** Writing – review & editing, Supervision. **Zhe Tian:** Writing – review & editing, Supervision. **Jiying Liu:** Writing – review & editing, Supervision, Methodology, Conceptualization, Project administration, Funding acquisition.

Declaration of competing interest

The authors declare that they have no known competing financial interests or personal relationships that could have appeared to influence the work reported in this paper.

Data availability

Data will be made available on request.

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